

Denne fil er downloadet fra Danmarks Tekniske Kulturarv

www.tekniskkulturarv.dk

Danmarks Tekniske Kulturarv drives af DTU Bibliotek og indeholder scannede bøger og fotografier fra bibliotekets historiske samling.

Rettigheder

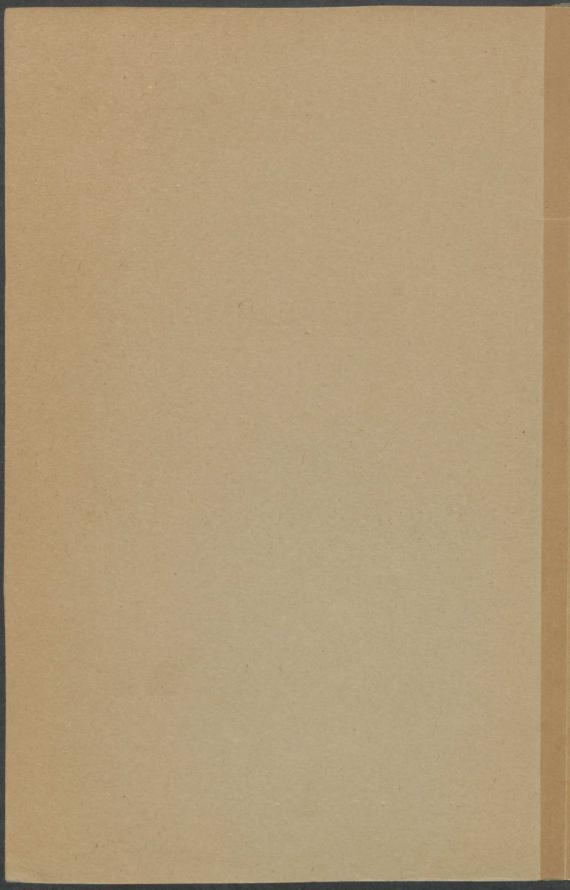
Du kan læse mere om, hvordan du må bruge filen, på www.tekniskkulturarv.dk/about

Er du i tvivl om brug af værker, bøger, fotografier og tekster fra siden, er du velkommen til at sende en mail til tekniskkulturarv@dtu.dk

THE SULLIVAN AIR LIFT PUMPING SYSTEM.

1917

T.B. 621,65-69 Sull.- 9





Sullivan Machinery Company

BULLETIN 71-C.

CHICAGO, MARCH, 1917

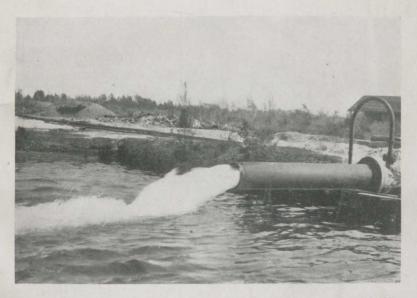
The Sullivan Air Lift Pumping System

Methods and Apparatus

UMPING water from deep wells by means of compressed air has been practiced for many years, with varying degrees of success and efficiency. The purpose of this bulletin is to call attention to the important advantages of air lift pumping, when properly applied, and to show how Sullivan engineering and Sullivan equipment secure a high degree of effectiveness in this field.

The air lift department of this company embodies a separate corps of engineers, whose efforts are devoted solely to problems relating to pneumatic pumping, and whose experience in this field embraces nearly twenty-five years of manufacture and installation.

Those interested in securing water supply from deep wells are urged to submit their requirements and conditions to this department.



Well No. 3, Prairie Pebble Phosphate Company, Mulberry, Florida, pumped by the Air Lift, and flowing 4600 gallons per minute

Copyright, 1917 SULLIVAN MACHINERY COMPANY

621.65-69 Sul-9

Standard Terms Used in Air Lift Work

The sketch on this page illustrates the terms regularly used in referring to air lift pumping, and these terms and their application should be thoroughly understood as a preliminary to study of the subject. They are explained more fully below.

I. STATIC HEAD: Normal water level when not pumping, measured from surface or top of well casing.

2. Drop: Point to which the water level drops below the static

head while being pumped.

3. Pumping Head: Level of water when pumping as compared to ground surface or top of well casing. Static Head + Drop = Pumping Head.

4. ELEVATION: Point above the ground surface or top of well casing to which

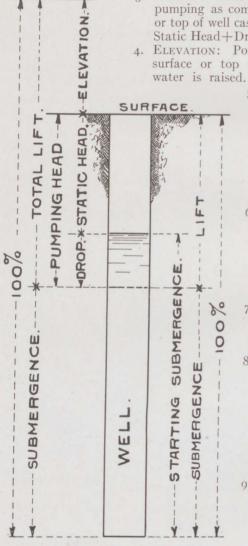
5. TOTAL LIFT: Distance water is elevated, from level when pumping, to point of discharge. at an elevation, and includes: Elevation+Static Head+Drop=Total Lift.

6. Lift: Distance the water is elevated from level when pumping to point of discharge at surface, and includes: Static head+Drop= Lift (or pumping head).

7. Submergence: Distance below the pumping head at which the air picks up the water.

8. 100 PER CENT: The vertical distance the air travels with the water from point introduced to point discharged, and includes: Total Lift or Lift+ Submergence = 100 per cent.

9. STARTING SUBMERG-ENCE: Distance below the static head at which the air picks up the water and includes:



Drop+Submergence = Starting Submergence.

Among the advantages of pneumatic pumping may be noted:

- I. QUANTITY: More water can be secured from the same wells than by any other system.
- 2. QUALITY: Improvement in the character of the water, due to aeration, as to purity and solubility.
- 3. Temperature: Reduction in temperature, due to absorption of the heat in the water, by the air.
- 4. Durability and Simplicity: There are no moving parts in the well.
- 5. The apparatus is always in order, and is not affected by mud, grit, floating sand or by long shut downs.
- 6. SUSTAINED EFFICIENCY.

These advantages may be explained more fully as follows:

There is no question but that more water can be secured from a deep well with the air lift than with any other method of pumping, provided the conditions are proper for its use. This is especially true where it is desired to increase the yield from a flowing well; as, by mixing the ascending column of water with a small amount of air, the column is lightened and the head against the inflowing water reduced without in any way retarding the flow.

The deep wells of industrial and public ownership are found to be remarkably free from disease germs, as the casing, driven down to hard pan above the gravel formation, or into rock, shuts off contamination from the surface.

Gravel beds in or near a river may be made the source of a pure water supply by drilling wells and casing them low enough so that the water will pull down through the sand and gravel. The erosion of the river keeps the top of the infiltration bed clean.

Aeration is acknowledged to be one of the principal methods for purifying water in filtration plants. If this occurs with air at low pressure, the perfect mixture of air and water in an air lift should and does cause much more complete purification.

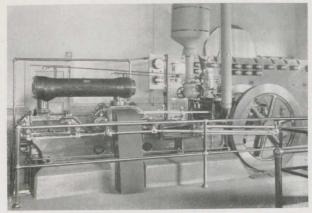
Free sulphur gas is encountered in many underground waters. This gas is almost entirely removed by the action of aeration in an air lift system, and the water from many sources of supply that is unfit for domestic use, on account of sulphur discoloration when pumped by a direct acting pump, is entirely freed from its odor and staining effects by the use of the air lift.

The aeration of water containing large quantities of iron causes a precipitation of this solid in the shape of a yellow mud, and while the freeing action is not as rapid as with sulphur, a large percentage of the iron can be eliminated by allowing time for settling.

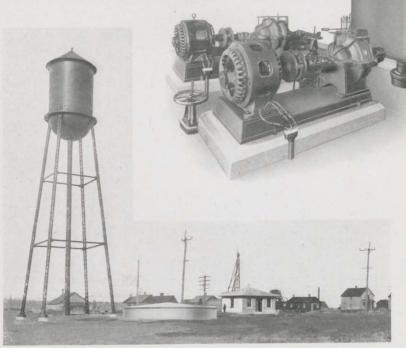
Advantages of the Air Lift

Quantity

Quality



Sullivan Air Lift Installation, Municipal Water Works, Buhl, Minnesota. The air Lift raises the water into a reservoir, from which it is forced into the standpipe or the mains by centrifugal pumps. The Compressor is a Sullivan "WB2," two-stage Machine



Most of the other solids contained in underground waters, if not thrown off altogether by aeration, are rendered easier of treatment. For instance, while practically as much scale will form in a boiler from water lifted with air as by other methods of pumping, yet the scale so formed will be more of the consistency of chalk than of the cement-like deposits of water not aerated.

As heat is driven from air by compression, through re-expansion it must absorb heat from any material with which it may come in

Temperature

contact. The expansion of the air in the eduction pipe of the air lift absorbs heat from the water and lowers its temperature. If the water is naturally cold, the reduction will be slight, but if the temperature

of the water is high, the reduction will be marked. This is a great advantage in water to be used for condensing purposes, as the reduction of a few degrees in the temperature of the water will make a great difference in the amount of coal used.

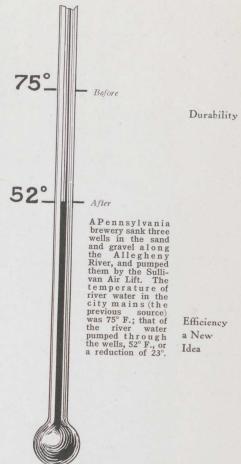
The air lift installation is more durable, and requires less attention and repairs than any other method of pumping. When once the well or wells are properly adjusted, the installation requires no further attention. The water never comes in contact with any moving parts and the machinery is in the power house directly under the eye of the engineer, avoiding all pulling of sucker rods, working barrels, et cetera.

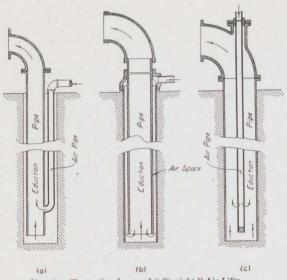
On account of the ease with which installations may be made, it is only in recent years that *efficiency* has been thought of in connection with air lift work.

As a general thing, manufacturers have sold compressors for this work upon the specifications of the customer, without making a thorough investigation of his requirements or

advising him as to the best method of developing his supply. It has been so easy to turn the air loose in a well and get certain results, and there has been so little known of the laws governing this method of pumping, that haphazard installations have given it a reputation for inefficiency that is not warranted, and which a careful study of the subject will tend to remove.

When Dr. Pohlé worked out his theory of a submergence of sixty per cent and of alternate plugs of air and water, he secured an efficiency of from twenty to twenty-five per cent, which at that time was thought to be as high as could be depended upon. Since then, experiments





Sketches illustrating forms of "Straight" Air Lifts

and careful investigation as to methods of mixing air and water, proper proportioning of piping, et cetera, have almost doubled this figure; and increasingly satisfactory results are being secured by careful study of the laws governing this work, by the care in installation and the improved methods and equipment used by this company.

0

A discharge consisting of alternate plugs of air and water is a common result of most forms of air lift in which open-end piping is used, and which is generally termed the "straight air" system. Although at times, a more or less constant discharge is secured, this is considered one of the vagaries of the system and the cause is not thoroughly understood.

There are three general systems of "straight air" pumping.

- I. THE OUTSIDE SYSTEM. (See "A" in the cut above) in which the air is carried down outside of the eduction pipe, into which it is connected a short distance above the bottom.
- 2. The Central Pipe System. (Sketch "C") in which the air is carried down through a pipe suspended inside the eduction pipe and allowed to discharge into the water through an open end.
- 3. The Reservoir System. (Sketch "B") in which an eduction pipe is suspended in a casing, allowing the air to pass down between the two and mix with the water at the bottom of the eduction pipe.

In all of these systems the principle is much the same. Pressure is built up in the air passage until it is sufficient to overcome the head

"Straight" Air Lift

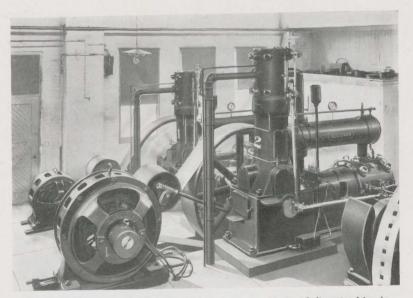
Types of "Straight" Air Lift due to submergence, when a large bubble of air passes into the eduction pipe. This flow of air from the pipe temporarily reduces the air pressure, through wire drawing; so that the weight of water in the well outside of the eduction and air pipes, which is due to submergence, shuts the air off and a plug of water follows the plug of air up into the eduction pipe, until the compressor has had time to build up the air pressure and another plug of air breaks through. This combat of the air and water pressure becomes rhythmic in its action, forming a succession of the latter than the submergence of the submergence of the latter than the submergence of the s

sion of air bubbles and water plugs in the eduction pipe, and is the cause of the "plugging" or unequal discharges found with this class of air lift installations.

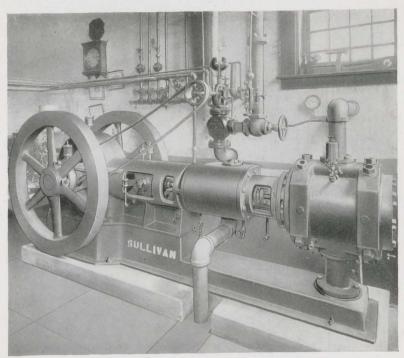
The theory of the air lift and the reason for submergence, are, that by mixing the water with air in the discharge, the water is made lighter, so that the pressure of the column of air and water in the bottom of the eduction pipe is less per square inch than that of the solid water



Maywood, Ill., City Water Works and one of the Sullivan Air Lift Boosters. There are two wells, pumped by Sullivan Foot Pieces, Boosters and Sullivan Angle Compound Air Compressors, see page 8. Flow, 1400 gallons per minute; lift, 250–300 feet. Elevated discharge from boosters, 20 feet.



Two Sullivan "WJ3" Angle Compound Air Compressors with short belt, motor drive, in City Water Works, Maywood, Illinois. (See page 7) No. 1 Compressor has a capacity of 628 cubic feet of free air per minute; No. 2, 886 feet.

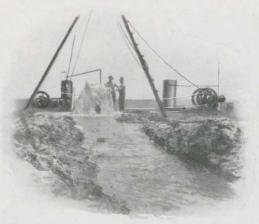


Sullivan "WA 4" Single-Stage, Steam Driven Air Compressor, pumping water at Fort Pitt Brewing Company's Plant, Sharpsburg, Pa.

outside, in the well, or in the rock, gravel or sand strata; and an upward flow is thus created.

The air lift should be a perfect expansion pump, but unfortunately there are other natural laws that take effect which prevent this in ordinary "straight air" installations.

The following are some of the conditions responsible for low efficiency in open end pipe air lifts.



An Open-Air Air Lift Plant, pumping water from a deep well for irrigation in New Mexico

1. The difference in pressure between the air and water where the former enters the eduction pipe is small, and the air flows into the water at a low velocity. In the open end pipe lift, the air must travel some distance towards the surface in the eduction line, before it can

expand sufficiently to form a plug and carry the water with it. This means a loss in submergence.

2. As the bubble of air travels towards the surface and the pressure above decreases, it must expand, and being confined in the pipe, can only do this in a vertical direction—resulting in increased velocity and friction and a greater displacement in the eduction pipe.

3. Owing to well known laws, the flow at the point of contact between a gas or liquid and the walls of the passage is retarded, and

water, being heavier than air, is retarded more by friction until it is pulled back around the bubble of air. This bubble becomes elongated and at times slips through, joining the preced-



Flow from a Sullivan Air Lift installation in the California Oil Fields

ing bubble and leaving the water behind. This slippage represents loss of efficiency and is manifested by variations in the plugging discharge.

Reasons for Low Efficiency



Drilling No. 9 Well, 16-in. in diameter, for Prairie Pebble Phosphate Co., Southard Contracting Company, Contractors



Well No. 10, and Receiving Basin, Mulberry, Florida

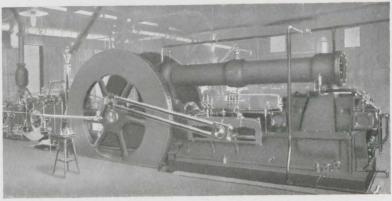
0

4. The air, like any confined gas, is always seeking a chance to escape, and any inequalities in the flow passage or abrupt changes in direction of flow, offer the opportunity sought, resulting in a still further slippage.

These disadvantages can be overcome: and the best results are obtained by the following methods:

I. Means should be provided to secure a complete mixture of the air and water, forming an emulsion at the point at which the air is injected into the water. Then each particular small bubble of air will start its lifting effect at once.

2. A Venturi or throat should be provided just above the mixer. This will increase the velocity, give a jet effect at this point, insure



Sullivan 2450-foot Tandem Corliss Air Compressor used for Air Lift Pumping in Plant of Prairie Pebble Phosphate Company

Requirements for Efficient Air Lift a more thorough mixture of the air and water, and exert a continuous upward pressure upon the ascending column.

3. The eduction pipe should be arranged to allow proper expansion of the air so far as possible and to prevent excessive velocity as the point of discharge is approached.

4. An absolutely smooth passage for the air and water is essential. Even the swirl caused by the recess in a coupling, occurring as often as it does in a long string of pipe, will cause a considerable loss.

5. Proper proportioning of air and water pipes for the work to be done is very important.

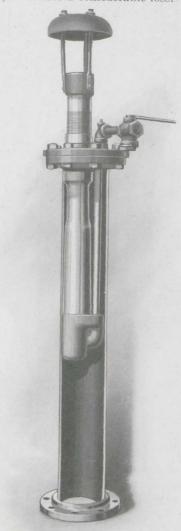
Detailed Description of the Sullivan Air Lift Pumps and Well Tops

The Sullivan Air Lift Pumps, described below and illustrated on pages II to 14, will be seen to embody exactly the principles expressed above; and the satisfaction they have given in actual practice under a wide range of working conditions, . bears out the claim made for them, that they comprise the most efficient apparatus vet devised for raising water by air lift methods. Tables of dimensions and weights appear on page 27.

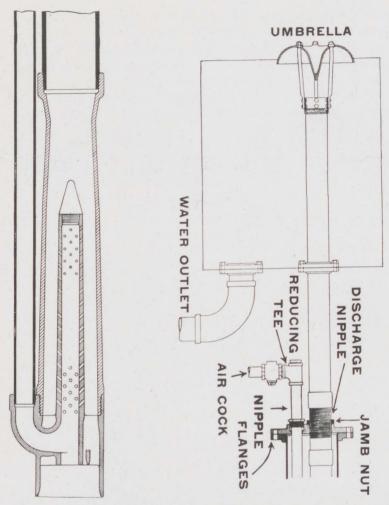
The Sullivan Standard Air Lift Foot Piece

(Outside Air Line)

The Sullivan Standard Pump or Foot Piece, illustrated on this page, is made



Sullivan Standard Foot Piece and Well Top, showing relative location of casing, flanges, et cetera



Sullivan Standard Air Lift Pump and Umbrella Well Top, with casing flanges and connections

of bronze, with an air passage, leading from the outside, terminating in a perforated vertical tube below a Venturi or throat, so that the air is broken up into small jets, insuring a perfect mixture, or emulsion, of the air and water, and increasing the velocity in the throat to produce an ejector effect. By having a large number of holes in the mixing tube, a mixture containing more or less air, depending upon the lift, is made automatically, without change in the pump. An opening just below the mixing tube permits sand or scale, lodging in the tube when pumping is stopped, to drop through and not plug the perforations.

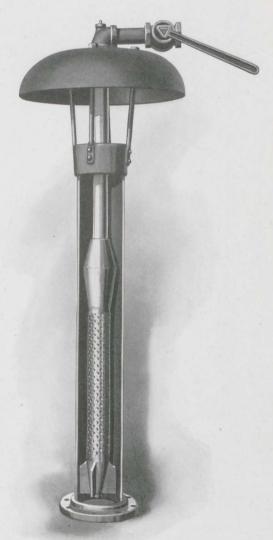
0

The Sullivan Standard Well Top

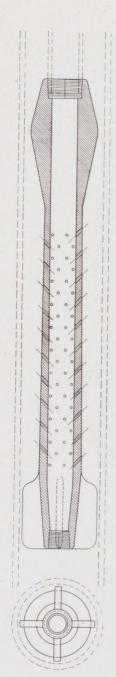
The Sullivan Standard Well Top (see pages 11 and 12) consists of flanges to seal the top of the casing, a water discharge nipple and umbrella separator, with jamb nut, and also with stuffing box for the air line, with air line nipple, back outlet elbow for gauge connection, and adjusting mine cock.

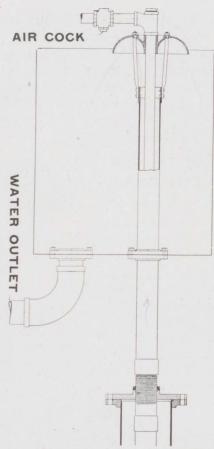
The Sullivan Central Air Lift Foot Piece

The Sullivan Central Pump or Foot Piece, illustrated on this page, is made of bronze, and consists of a perforated tube. It is suspended from the central air pipe, below a throat, and has wings to center it in the discharge pipe. The air is broken up into small jets by the perforations, insuring a perfect mixture, or emulsion, of the air and water, and increasing the velocity in the throat to produce an ejector effect. By having a large number of holes in the mixing tube, the mixture is automatically made to contain more or less air, depending upon the lift, without change



Sullivan Central Foot Piece and Umbrella Well Top; the flange shown at the foot of this cut and at the foot of the cut on page 11, is not a part of the equipment



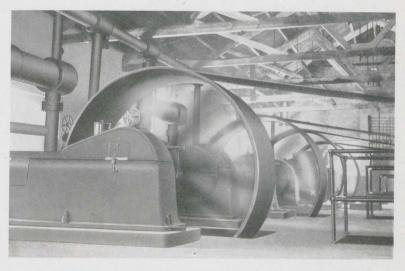


Sullivan Central Air Lift Pump and Umbrella Well Top, with casing flanges and connections. Shaded parts with air cock, tee and nipple are all that are furnished. Pipe, reservoir, etc., are to be provided by the customer

in the pump. An opening just below the mixing tube permits the sand or scale to drop through and not choke up the perforations when pumping is stopped. This foot piece, being suspended upon the central air line, can be raised or lowered without changing the discharge pipe.

The Sullivan Central Well Top

The well top for the central pump or foot piece, as shown here and also on page 13, consists of flanges to seal the top of the casing, a water discharge nipple with jamb nut, and an umbrella sepa-



Sullivan Belted Air Compressors operating Air Lifts for mill service at a well-known mine in Mexico

rator, having an opening in the top for suspending the air line; and is equipped with stuffing box, nipple, back outlet elbow for gauge connections and regulating mine cock.

NOTE: This head may also be made up with a back outlet elbow on the discharge for suspending the air line, when it is necessary to carry the discharge to an elevation.

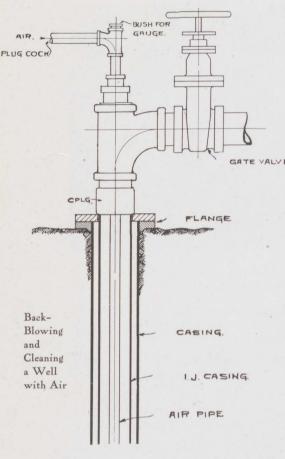
Ratio of Lift to Submergence

In actual practice, it has been found that the submergence may be varied with the lift, shorter lifts requiring a greater percentage of submergence. While there is no definite division line, the following proportion of submergence to lift will be found effective:

| For lifts up t | 50 feet70 to 66 per cent submergence |
|----------------|--|
| 50 t | 100 feet |
| 100 t | 200 feet 55 to 50 per cent submergence |
| 200 to | 300 feet 50 to 43 per cent submergence |
| 300 to | 400 feet 43 to 40 per cent submergence |
| 400 to | 500 feet 40 to 33 per cent submergence |

It will be readily understood that the diameter of the water discharge or eduction pipe is of prime importance to secure the best efficiency, and must bear a relation not only to the amount of water to be handled, but also to the amount of air; so that for an equal amount of water the pipe size may vary with the lift and also with

Proportioning of Piping



Arrangement of piping a well for back-blowing

the percentage of submergence, as both factors change the amount of air. If the pipe is too large, then there is slippage of the air past the water, unless more air is used to keep up the velocity; or if too small a pipe is used, undue friction will increase

Each separate well or group of wells is an engineering problem that should be carefully studied, and the most complete data obtainable should be furnished, so that a system can be designed to cover all points.

The standard practice of well drillers is to equip gravel and sand wells with a strainer, designed to shut out the sand from the working barrel of a deep well pump. In time, these strainers become clogged with sand and the flow into the well is thus reduced. By a system of back-blowing, the output from such wells can be permanently increased.

The correct strainer for wells of this class, pumped by the air lift, is a perforated screen with openings of a suitable size to admit the fine material into the well, from which it can be pumped, and to hold back the coarser particles, so as to form a natural gravel filter bed outside of the artificial one.

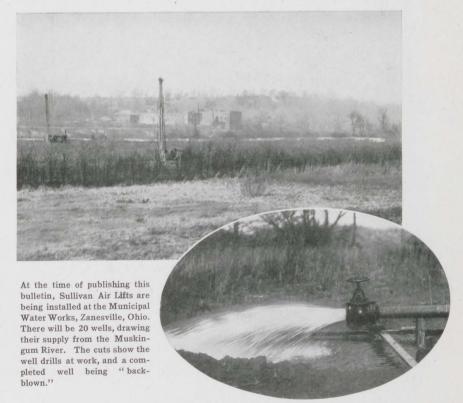
The force available for getting water into a well is the head, due to the difference between the static level in the water strata outside the well and the pumping level in the well, minus friction due to the strata and screen. Therefore, the more this friction can be reduced, the greater will be the flow, providing of course, that an abundance of water is available.

"Back-blowing" can be applied to all wells. The top of the well casing should be sealed; next, by closing the discharge pipe while

the air lift is in operation, the air will be forced through the foot piece and will drive the water ahead of it out through the strainer and float the finer sand. Then, by opening the discharge, the flow will resume its course toward the surface and bring a portion of the floating sand with it. By a repetition of this operation and by increasing the back pressure if necessary, all of the fine sand immediately outside of the strainer will be drawn into the well and discharged at the surface, and the coarser gravel will be collected outside of the screen in such quantities as to shut off the sand and increase the flow into the well, without changing the piping in the well. This process may be repeated at any time, so that the screen and adjacent strata can be kept clear.

When wells are drilled in rock, the action of the drilling tool forces the cuttings back into the crevices in the rock. These may be loosened and pumped out of the well in the same manner.

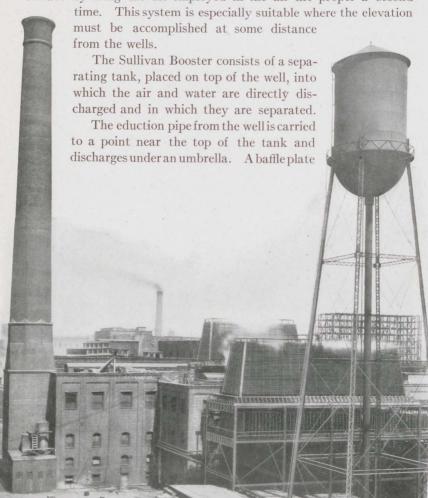
In the case of wells in fine material and quicksand it is often possible to set a strainer in the sand and drill auxiliary holes alongside the well down to the top of the strainer. Then foreign gravel may be



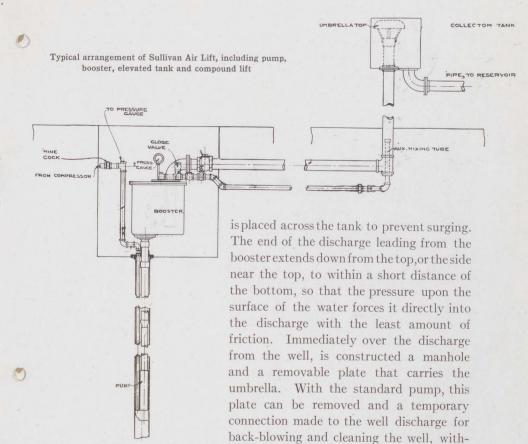
dropped down, which will roll in alongside of the screen and take the place of the sand pumped out—often increasing the yield four-fold, by affording, outside of the gravel bed, a larger area through which the water may leave the sand.

The Sullivan Booster System for Elevated Discharge

By means of the Sullivan "Booster" System (illustrated on pages 19 and following), the water may be lifted to an elevation above the surface by using the air employed in the air lift proper a second



Power house and storage tank of Morris & Company, Union Stock Yards, Chicago

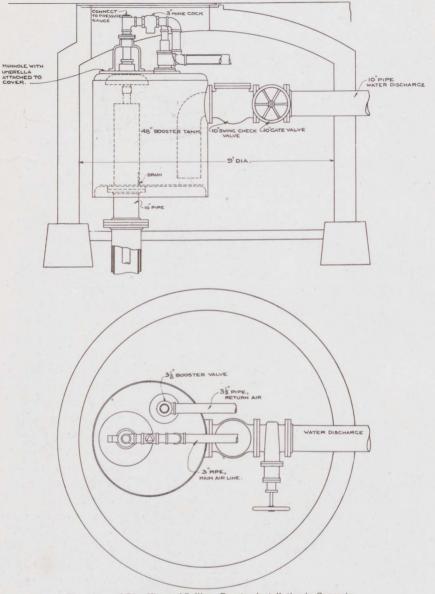


out removing the booster, keeping it free from sand or drillings.

The Central Pump and air line, when this installation is used, are installed through this opening after the booster is in place and suspended in the eduction pipe by means of the manhole plate, which affords an easy means of raising or lowering the pump without disturbing any other part of the equipment.

The vent valve is a circular slide valve made of bronze and operated by the water in the booster, through the use of a diving bell, non-leaking float and rod; and is adjusted from the outside by means of a spring under a cap in order to maintain the necessary pressure to force the water to the required head.

The principle involved in the operation of the booster is as follows: The velocity of water and air in an air lift is always greatest as it nears the point of discharge. This is demonstrated by the force of the discharge into the atmosphere. By discharging into a closed tank, the kinetic energy of the column of water and air under arrested



Elevation and Plan Views of Sullivan Booster installation in Concrete Chamber, with Piping Connections

motion is used to recompress the air and a certain amount of the power originally expended is returned.

Compressed air under ordinary air lift conditions is wasteful if used to force water horizontally, as the air, being lighter than water, goes to the top of the pipe and fails to bring the water with it.

With the use of a booster, the air does not follow the water but is held in the tank, building up pressure to force the solid body of water through the horizontal and vertical lines.

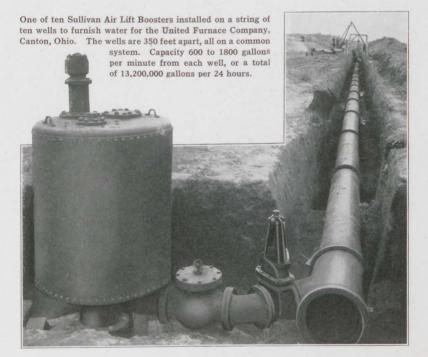
The booster is usually set in a pit below the ground level and enclosed in a brick or concrete dry well, but may be located at any convenient point, as provision is made for automatic drainage back into the well when pumping is stopped.

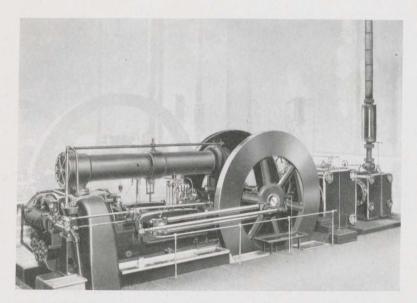
It will readily be seen, therefore, that with this arrangement, any number of wells may discharge into a common delivery pipe. This constitutes a great advantage when the elevation to be reached is at some distance from a well or group of wells. A check valve should be installed in the discharge line immediately outside of the booster.

The engineer operates the complete plant from the compressor, no change being required after the adjustment has once been made. Varying the speed of the compressor secures a greater or less amount of water.

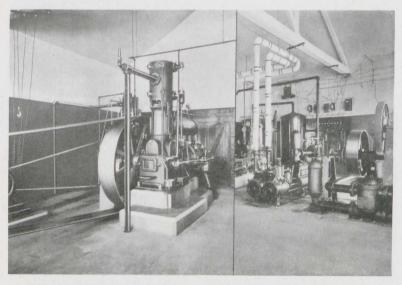
There are three ways in which the air escaping from the booster may be employed.

- 1. It may be allowed to escape through the vent valve to the atmosphere.
 - 2. It may be returned to the intake of the compressor. The

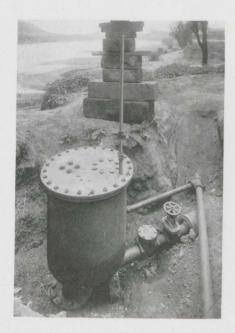




Sullivan "WC" Tandem Corliss Compound Two Stage Air Compressor used for Air Lift pumping at packing house of Morris & Company, Chicago. (See page 18.) This Compressor has a rated capacity of 2450 cubic feet of free air per minute



Sullivan "WJ3," Angle-Compound Air Compressor in the Municipal Water Works at Dickinson, North Dakota



Sullivan Booster on one of three wells at Sharpsburg, Pa. Horizontal discharge line 1350 ft. long; final elevation, 25 ft.

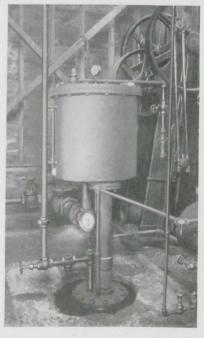
The accompanying illustrations and notes show some of the installations of this character that have been made by this company. More detailed reports on these and many other similar booster plants are also available.

The Air Lift for Acids and Chemicals

The air lift is admirably adapted for pumping acids, slimes, pulps and other solutions containing chemicals which rapidly eat out the working parts of mechanical pumps. For service in connection with Pachuca tanks, flotation and other wet ore dressing systems, the air lift is very desirable. Advantages secured in handling such liquids by this method, as embodied in Sullivan practice, include

fact that this return air is at a lower temperature than atmospheric air and at a slightly greater pressure, secures a decided gain in volumetric efficiency.

3. Where the water is discharged at an elevation above the surface, either near or at some distance from the well or wells, a greater gain is secured by discharging the air from the booster or boosters through a mixing tube in the base of the riser pipe, lightening the water discharge column and reducing the head.



Sullivan Booster on a well at Rubsam and Hormann Brewing Co., Staten Island. N.Y.; discarded deep well pump in background. This plant gives 55 gallons per minute instead of 30 gallons, (capacity of old pump) and requires only 13¾ H. P. as compared with 20. At the same plant, a Sullivan Air Lift Booster was substituted for an Air Lift of another system. The flow in gallons per minute was increased from 50 to 100, with 20 per cent less air.



At Fort Worth, Texas, the Missouri, Kansas & Texas R. R. shops installed a Sullivan Air Lift Booster and Compound Air Lift Pump (shown in photo). Depth of pump in well, 545 ft.

Height of tank from surface, 75 ft.

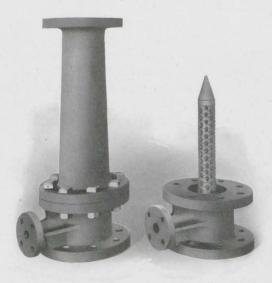
Air pressure to raise water to surface, 80 lbs.

Back pressure in booster to raise water from ground to top of tank, 10 lbs.

Flow of water per hour, 6000 gals.

(1) length of service, (2) absence of repairs, (3) cheapness of renewals, (4) variation in capacity; to which may be added (5) high efficiency on lifts of any height and of varying submergence, (6) solutions carried horizontally as well as vertically, (7) any number of lifts handled from a central power plant, (8) simplicity and sustained effectiveness, (9) apparatus furnished in any materials necessary to resist chemical action.

This company has installed numerous pumps for service such as is described above, made of various acid-resistant materials, such as hard or soft lead, cast iron or steel, bronze, rubber, etc. Sullivan Acid Pumps are designed to work satisfactorily under low submergence. The following data should be furnished in order to permit estimates to be submitted.



Sullivan Air Lift Acid Pump, supplied as desired in cast iron or cast steel

- 1. Temperature of liquid.
- 2. Specific gravity of liquid.
- 3. Material of which the pump is to be made.
- 4. Kind of acid to be pumped.
 - 5. Lift in feet.
- 6. Submergence in feet.
- 7. Pounds to be pumped per minute.

The question of an air compressor for air lift work is one that should receive careful atten-

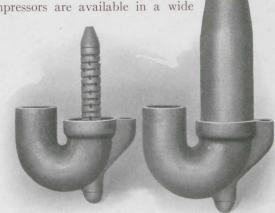
tion, because the efficiency of the machine, under this class of work, is immediately demonstrated. When compressed air is used for manufacturing purposes, it is frequently difficult to check up the

amount of work done, but with the air lift, the foot pounds of work done are easily measured and comparisons made. The reliability and economy of the compressor should therefore receive the close attention of the engineer.

Sullivan air compressors are available in a wide

range of styles, pressures, capacities and for operation by steam, belt, electric motor, et cetera.

Some of these styles are shown in the accompanying illustrations. Bulletins on Sullivan air compressors will be furnished on request.



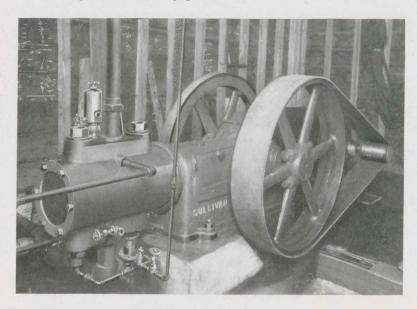
Sullivan Air Lift Acid Pump, supplied in hard or soft lead, or other acid resisting materials



Separating Head for Acid Pumping

In the latter pages of this bulletin will be found tables of pipe sizes and specifications, air compressor data and other reference material, that are required in air lift calculations. It is hoped that their publication here may prove of assistance to intending purchasers.

Note. This company reserves the right to alter its machines and apparatus, and to furnish them, when so altered, without reference to the illustrations or reading matter contained in the present bulletin.



Sullivan "WG-3" Single Stage, Belted Air Compressor operating the Sullivan Air Lift Booster shown on page 23. This plant has since been enlarged by the addition of a Sullivan Angle-Compound Compressor of 445 feet capacity.

Sullivan Standard Air Lift Pumps

General Code Word, Obarma

| Size Discharge, | Approx. Capacity | Diameter Well | Approx. Shipping | Code Word for |
|---|---|---|--|---|
| Inches | Gallons per Min. | Head, Inches | Weight, Pounds | Complete Pump |
| I I 1 1/4 I 1/2 2 2 1/2 3 3 1/2 4 4 1/2 5 6 7 | 5 10 20 50 75 100-150 150-200 200-300 300-350 350-500 500-800 | 4 4 4 6 6 6 8 8 8 10 10 | 125 135 150 175 185 200 250 300 | Obermans Obarmare Obarmati Obarmes Obarment Obarmenur Obarmoi Obba Obbarum Obbis Obdidi |



High Lift Pachuca Tanks at a Mexican Mine Solutions handled by the Air Lift.

Sullivan Central Air Lift Pumps

General Code Word, Obdormio

| Sullivan | Boosters | |
|--------------|-------------|----|
| General Code | Word, Oblea | to |

| Size Discharge, | Approx. Capacity | Diameter Well | Approx. Shipping | Code Word for |
|---|---|---|--|---|
| Inches | Gallons per Min. | Head, Inches | Weight, Pounds | Complete Pump |
| 1 11/4 11/2 2 21/2 3 3 1/2 4 41/2 5 6 7 8 9 10 11 12 13 14 15 | Capacity determined by size of air line and effective discharge area. | 2 3 3 4 4 5 6 6 7 8 9 10 11 12 13 14 16 18 18 | 25 30 35 40 45 50 55 60 65 75 100 125 150 225 225 225 225 225 300 325 | Obduca Obducatur Obducetr Obducets Obducits Obducits Obducitus Obducitus Obducitus Obducitus Obediens Obedielur Obedio Obedivist Obedivist Obeor Obequito Obesitae Obesis Obesis Obesum |

| Diameter, Inches | Height, Inches | Capacity Gallons, per Minute | Approx. Shipping Veight, Pounds | Code Word for Booster Complete |
|----------------------------|----------------------------|--|------------------------------------|--|
| 24 30 36 42 48 | 24 30 36 42 48 | 50-100 100-200 200-400 400-600 600-800 | 1500 2000 1500 | Oblidere Oblidetus Oblidis Oblidunt Oblisi |

Boosters will be made of cast iron up to the 36-inch size; larger, of boiler plate, riveted.

Inlet and discharge flanges are furnished to suit specifications.

EQUIPMENT.—The prices of complete pumps include the Foot Piece, Companion Flange, Made-up Flange, Bolts and Gasket, Umbrella Separator or Long Sweep Elbow, Nipples, Gauge Connection and Mine-Cock Adjusting Valve. With the Standard and Central Pumps are also furnished a Long

Threaded Discharge Nipple with Jamb Nut and a Back Outlet Elbow. With the Standard Pump only, one Air Line Stuffing Box. When well casing is used as discharge with the Central pump, no flanges will be supplied.

Pipe Tables, for Reference*

Standard Pipe-Black and Galvanized All Weights and Dimensions are Nominal

| | Size | 24.82 | 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 | 37/2 33/2 | 44 472 | N 00 00 | 0 0 0 11 | 112 113 114 | 15 |
|-----------------|-----------------------|-------------------------------|---------------------------------------|----------------------------------|--------------------------------------|--------------------------------------|--------------------------------------|--------------------------------------|--------|
| Diameters | External | .405 | 1.050 | 2.375 | 4.500 5.000 5.563 6.625 | 7.625 8.625 8.625 9.625 | 10.750 10.750 10.750 11.750 | 12.750 12.750 14.000 15.000 | 16.000 |
| eters | Internal | .364 .493 .622 | .824 1.049 1.380 1.610 | 2.067 2.469 3.068 3.548 | 4.026 4.506 5.047 6.065 | 7.023 8.071 7.981 8.941 | 10.192 10.136 10.020 11.000 | 12.000 12.000 13.250 14.250 | 15.250 |
| S | Thickness | 880. | .113 | .154 .203 .216 .226 | .237 .258 .258 | .301 .322 .342 | .279 .307 .365 .375 | .330 .375 .375 .375 | .375 |
| Weight | Plain ends | .244 .424 .567 .850 | 1 130 1.678 2.272 2 717 | 3.652 5.793 7.575 9.109 | 12.538 14.617 18.974 | 23.544 24.696 28.554 33.907 | 31 201 34.240 40.483 45.557 | 43.773 49.562 54.568 58.573 | 62.579 |
| Weight per foot | Threads and couplings | . 245 . 568 . 852 | 1.134 1.684 2.281 2.731 | 3.678 5.819 7.616 9.202 | 10.889 12.642 14.810 19.185 | 23 769 25.000 28 809 34.188 | 32.000 35.000 41.132 46.247 | 45.000 50.706 55.824 60.375 | 64 500 |
| loni | Threads per | 27 18 18 14 | 11 11/2 11/2 11/2 | 111/2 8 8 8 | 00 00 00 00 | 00 00 00 00 | ∞ ∞ ∞ ∞ | ∞ ∞ ∞ ∞ | 00 |
| S | Diameter | .562 .685 .848 1.024 | 1.281 1.576 1.950 2 218 | 2.760 3.276 3.948 4.591 | 5.091 5.591 6.296 7.358 | 8.358 9.358 9.358 10.358 | 11.721 11.721 11.721 12.721 | 13.958 13.958 15.208 16.446 | 17 446 |
| Couplings | Length | 1,8 11,8 13,8 | 178 178 218 238 288 | 25/8 27/8 37/8 35/8 | 358 358 418 418 | 478 458 458 578 | 61/8 61/8 61/8 61/8 | 61/8 61/8 61/8 61/8 | 616 |
| | Meight | 029 | .343 | 1.208 1.720 2.498 4 241 | 4.741 5.241 8.091 9.554 | 10.932 13.905 13.905 17.236 | 29.877 29.877 29.877 32.550 | 43.098 43.098 47.152 59.493 | 63.204 |

The permissible variation in weight is 5 per cent above and 5 per cent below. Furnished with threads and couplings and in random lengths unless otherwise ordered to the season of the s

All Weights and Dimensions are Nominal Drive Pipe

| S | Weight | 2.380 3.748 4.493 5.973 | 6 740 7.439 II.871 I3.956 | 15.955 24.343 24.343 31.320 | 27.035 40.108 40.108 40.108 | 43.664 47.220 47.220 66.024 | 70 533 75 043 91.746 109 669 121.298 |
|-----------|-----------------------|----------------------------------|--------------------------------------|--------------------------------------|--|--------------------------------------|--|
| Couplings | Length | 358 418 418 418 | 41/8 41/8 51/8 51/8 | 51/8 61/8 61/8 61/8 | \$50 \$50 \$50 \$50 \$50 \$50 \$50 \$50 \$50 \$50 | 958 958 778 | 71/8 |
| S | Diameter | 2.923 3.486 4.111 4.723 | 5.223 5.723 6.410 | 8.474 9.588 9.882 9.882 | 10.588 11.950 11.950 11.950 | 12 950 13.950 13 950 15 438 | 16.438 17.438 18.675 19.913 21.913 |
| dəni. | Threads per | 8 8 8 8 | ∞ ∞ ∞ ∞ | ∞ ∞ ∞ ∞ | 00 00 00 00 | 00 00 00 00 | ∞ ∞ ∞ ∞ ∞ |
| per foot | Threads and couplings | 3.730 5.906 7.705 9.294 | 10.995 12.758 14.989 19.408 | 24.021 25.495 29.303 32.334 | 34.711 32 631 35 628 41.785 | 46 953 45.358 51 067 56 849 | 61.005 65 161 73.000 90 000 |
| Weight 1 | Plain ends | 3 652 5 793 7.575 9.109 | 10 790 12 538 14 617 18.974 | 23.544 24 696 28.554 31 270 | 33 907 31.201 34.240 40.483 | 45.557 43.773 49.562 54.568 | 58 573 62.579 60 704 76 840 85 577 |
| SS | Thickne | .203 | .237 .247 .258 .280 | .301 .322 .354 | .342 .279 .307 | .335 .330 .375 .375 | 375 393 409 409 |
| eters | Internal | 2.067 2.469 3.068 3.548 | 4.026 4.506 5.047 6.065 | 7.023 8 071 7.981 7.917 | 8.941 IO.192 IO.136 IO.020 | 12.000 | 14.250 15.250 16.214 17.182 19.182 |
| Diameters | External | 2.375 | 5.000 | 7 625 8 625 8 625 8 625 | 9 625 10 750 10 750 10 750 | 12.750 12.750 12.750 14.000 | 2 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 |
| | Size | 2 3/2 3 3/2 | 4 4 1/2 5 6 | r∞∞∞ | 0000 | 11221 | 14 17 17 18 0.D. 20 20 0.D. |

The permissible variation in weight is 5 per cent above and 5 per cent below. Furnished with threads and couplings and in random lengths unless otherwise ordered.

to so inches.

The weight per foot of pipe with threads and couplings is based on a length of Societ, including the coupling, but shipping lengths of small sizes will usually average less than so feast han one weight. All dimensions given in inches. On sizes made in more than one weight, weight desired must be specified. Taper of threads is % inch from 2 inches to 5 inches, and % inch from 6 inches

*(From Book of Standards, National Tube Co.)

All Weights and Dimensions are Nominal

| | Diam | eters | | Weight | | Join | nt |
|-------|----------|----------|----------------|---------------------------|---------------------|-----------------|-------------------------|
| Size | External | Internal | Thick- ness | per foot plain ends | Threads per inch | Length of Joint | Diameter of joint " D " |
| - | 2.050 | 2.050 | .100 | 2.206 | 14 | .967 | 2.340 |
| 21/4 | 2.250 | 2.284 | .108 | 2.759 | 14 | .992 | 2.606 |
| 21/2 | 2.750 | 2.524 | .113 | 3.182 | 14 | 1.017 | 2.866 |
| 23/4 | 3.000 | 2.768 | .116 | 3.572 | 14 | 1.042 | 3.122 |
| 3 | 3.250 | 3.010 | .120 | 4.011 | 14 | 1.067 | 5.380 |
| 31/4 | 3.500 | 3.250 | .125 | 4.505 | 14 | 1.092 | 3.640 |
| 31/2 | 3.750 | 3.492 | .120 | 4.988 | 14 | 1.117 | 3.898 |
| 38/4 | 4.000 | 3.732 | .134 | 5.532 | 14 | 1.142 | 4.158 |
| 4 | 4.250 | 3.974 | .138 | 6.060 | 14 | 1.167 | 4.416 |
| 41/4 | 4.500 | 4.216 | .142 | 6.600 | 14 | 1.192 | 4 674 |
| 41/2 | 4.750 | 4.460 | .145 | 7.131 | 14 | 1.217 | 4.930 |
| 43/4 | 5.000 | 4.696 | .152 | 7.870 | 14 | 1.242 | 5.194 |
| 5 | 5.250 | 4.944 | .153 | 8.328 | 14 | 1.267 | 5.446 |
| 53/18 | 5.500 | 5.102 | .154 | 8.792 | 14 | 1.292 | 5.698 |
| 55/8 | 6.000 | 5.672 | .164 | 10.222 | 14 | 1.342 | 6.218 |
| 55/8 | 6.000 | 5.620 | .190 | 11.789 | 111/2 | 1.373 | 6.246 |
| 61/4 | 6.625 | 6.287 | .169 | 11.652 | 14 | 1.405 | 6.853 |
| 65% | 7.000 | 6.652 | .174 | 12.685 | 14 | 1.442 | 7.238 |
| 71/4 | 7.625 | 7.263 | .181 | 14.390 | 14 | 1.505 | 7.877 |
| 75/8 | 8.000 | 7.628 | .186 | 15.522 | 111/2 | 1.573 | 8.238 |
| 814 | 8.625 | 8 249 | .188 | 16.940 | 111/2 | 1.636 | 8.867 |
| 856 | 0.000 | 8.608 | .196 | 18.429 | 111/2 | 1.673 | 9.258 |
| 95% | 10.000 | 9.582 | .209 | 21.855 | 111/2 | 1.773 | 10.284 |
| 105/8 | 11.000 | 10.552 | .224 | 25.780 | 111/2 | 1.873 | 11.314 |
| 115% | 12.000 | 11.514 | .243 | 30.512 | 111/2 | 1.973 | 12.352 |
| 121/2 | 13.000 | 12:482 | .259 | 35.243 | 111/2 | 2.073 | 13.384 |
| 131/2 | 14.000 | 13.448 | .276 | 40.454 | 111/2 | 2.173 | 14.418 |
| 141/2 | 15 000 | 14.418 | .291 | 45.714 | 111/2 | 2.273 | 15.440 |
| 1516 | 16 000 | 15.396 | .302 | 50 632 | 111/2 | 2.373 | 16.47 |

The permissible variation in weight is 5 per cent above and 5 per cent below. Furnished in random lengths unless otherwise ordered.

Furnished in random lengths unless otherwise objects. Regular taper of threads is % inch diameter per foot length for all sizes, but will furnish ¼ inch, ½ inch, or % inch taper if so ordered.

All weights given in pounds. All dimensions given in inches.

All weights given in pounds. An undershows given in death of on sizes made in more than one weight or thread, weight and number of threads desired must be specified.

Thickness of walls make it impracticable to cut threads of coarser pitch than

shown on table.

Decimal Parts of an Inch*

Decimals of an Inch for Each 1/64th

| 1/32 | 364 | Decimal | Fraction | 1/32 | 1/64 | Decimal | Fraction |
|------|-----|----------|----------|------|------|----------|----------|
| | I | .015625 | | 1 | 33 | . 515625 | |
| I | 2 | .03125 | | 17 | 34 | .53125 | |
| * | 3 | .046875 | | | 35 | .546875 | |
| 2 | 4 | .0625 | 1/16 | 18 | 36 | . 5625 | 916 |
| | 5 | .078125 | | | 37 | . 578125 | |
| 3 | 5 6 | .09375 | | 19 | 38 | - 59375 | |
| | 7 8 | .109375 | | | 39 | .609375 | |
| 4 | 8 | .125 | 1/8 | 20 | 40 | .625 | 5% |
| | 9 | . 140625 | | | 41 | .640625 | |
| 5 | 10 | . 15625 | 1 | 21 | 42 | .65625 | - |
| | II | .171875 | | | 43 | .671875 | |
| 6 | 12 | . 1875 | 310 | 22 | 44 | .6875 | 11/18 |
| | 13 | . 203125 | | | 45 | .703125 | |
| 7 | 14 | .21875 | | 23 | 46 | .71875 | |
| | 15 | . 234375 | | | 47 | .734375 | 97 |
| 8 | 16 | .25 | 34 | -24 | 48 | .75 | 84 |
| | 17 | . 265625 | | | 49 | .765625 | |
| 9 | 18 | . 28125 | | 25 | 50 | .78125 | 1 |
| | 19 | . 296875 | | | 51 | .796875 | 12/ |
| IO | 20 | .3125 | 518 | 26 | 52 | .8125 | 1316 |
| | 21 | .328125 | | | 53 | .828125 | |
| II | 22 | .34375 | | 27 | 54 | .84375 | |
| | 23 | -359375 | | | 55 | .859375 | 7% |
| 12 | 24 | , .375 | 3/8 | 28 | 56 | .875 | 18 |
| | 25 | .390625 | | | 57 | .890625 | |
| 13 | 26 | .40625 | | 29 | 58 | .90625 | |
| | 27 | .421875 | | | 59 | .921875 | 10/ |
| 14 | 28 | -4375 | 7/18 | 30 | 60 | -9375 | 16/18 |
| | 29 | .453125 | | | 61 | .953125 | |
| 15 | 30 | .46875 | | 31 | 62 | .96875 | |
| | 31 | 484375 | | | 63 | .984375 | |
| 16 | 32 | .5 | 1/2 | 32 | 64 | I | I |

Mechanical, Electrical and Heat Equivalents*

One Horse Power is equal to: 746 Watts, 0.746 K. W., 33,000 ft.-lbs. per minute, 550 ft.-lbs. per second, 2,545 heat-units per hour, 42.4 heat-units per minute, 0.707 heat-units per second, 0.175 lbs. carbon oxidized per hour, 2.64 lbs. water evaporated per hour from and at 212° F.

^{*(}From Book of Standards, National Tube Co.)

Contents in Cu. Ft. and U. S. Gallons of Pipes and Cylinders of various inside diameters and one foot in length.*

(r gallon = 23r cubic inches. r cubic foot = 7.4805 gallons.)

| er . | For I ft. | in length | | For I ft. | in length | h | For I ft. | in length |
|-------------------------------------|---|---|---------------------------------|--|---|-------------------------------|--|---|
| Diameter in inches | Cubic feet,also area in square feet | U.S. gallons | Diameter in inches | Cubic feet, also area in square feet | U.S. gallons | Diameter in inches | Cubic feet, also area in square feet | U.S. gallons |
| 5/16 3/8 7/16 1/2 | .0005 .0008 .0010 | .0025 .0040 .0057 .0078 | 6% 7 71/4 71/2 78/4 | .2485 .2673 .2867 .3068 .3276 | 1.859 1.999 2.145 2.295 2.450 | 19 19½ 20 20½ 21 | 1.969 2.074 2.182 2.292 2.405 | 14.73 15.51 16.32 17.15 17.00 |
| 916 5/8 11/16 3/4 13/16 | .0017 .0021 .0026 .0031 | .0129 .0159 .0193 .0230 | 8 81/4 81/2 83/4 | .3491 .3712 .3941 .4176 | 2.611 2.777 2.948 3.125 | 21½ 22 22½ 22½ 23 | 2.521 2.640 2.761 2.885 | 18.86 19.75 20.66 21.58 |
| 75 1516 1 114 | .0036 .0042 .0048 .0055 .0085 | .0269 .0312 .0359 .0408 .0638 | 9 954 952 984 10 | .4418 .4067 .4922 .5185 | 3.305 3.491 3.682 3.879 | 23½ 24 25 26 | 3.012 3.142 3.409 3.687 | 22.53 23.50 25.50 27.58 |
| 11/2 13/4 2 21/4 | .0123 .0167 .0218 .0276 | .0918 .1249 .1632 .2066 | 101/4 101/2 108/4 11 | .5454 .5730 .6013 .6303 .6600 | 4.080 4.286 4.498 4.715 4.937 | 27 28 29 30 31 | 3.976 4.276 4.587 4.909 5.241 | 29.74 31.99 34.31 36.72 39.21 |
| 21/2 23/4 3 31/4 31/2 | .0341 .0412 .0491 .0576 .0608 | .2550 .3085 .3072 .4309 | 111/4 111/2 113/4 12 | .6903 .7213 .7530 .7854 | 5.164 5.396 5.633 5.875 | 32 33 34 35 | 5.585 5.940 6.305 6.681 | 41.78 44.43 47.16 49.08 |
| 38/4 4 41/4 43/2 | .0767 .0873 .0985 | .4998 .5738 .6528 .7369 .8263 | 12½ 13 13½ 14 14½ | .8522 .9218 .9940 I .069 I .147 | 6.375 6.895 7.436 7.997 | 36. 37 38 39 | 7.069 7.467 7.876 8.296 | 52.88 55.86 58.92 62.06 |
| 43/4 5 51/4 51/2 | .1231 .1364 .1503 .1650 | .9206 1.020 1.125 1.234 | 15 15½ 16 16½ | 1.227 1.310 1.396 1.485 | 8.578 9.180 9.801 10.44 11.11 | 40 41 42 43 44 | 8.727 9.168 9.621 10.085 | 65.28 68.58 71.97 75.44 |
| 53/4 6 01/4 61/2 | .1803 .1963 .2131 .2304 | I.349 I.469 I.594 I.724 | 17 17½ 18 18½ | I.576 I.670 I.767 I.867 | 11.79 12.49 13.22 13.96 | 45 46 47 48 | 10.559 11.045 11.541 12.048 12.566 | 78.99 82.62 86.33 90.13 94.00 |

To find the capacity of pipes greater than the largest given in the table, look in the table for a pipe of one-half the given size, and multiply its capacity by 4; or one of one-third its size, and multiply its capacity by 0, etc.

To find the weight of water in any of the given sizes, multiply the capacity in cubic feet, by 62½ or the capacity in gallons by 8½, or, if a more accurate result is required, by the weight of a cubic foot of water at the actual temperature in the pipe.

Given the dimensions of a cylinder in inches, to find its capacity in U. S. gallons: Square the diameter, multiply by the length and by 0.0034. If d = diameter, l = length, gallons = $\frac{d^2 \times 0.7854 \times l}{231} = 0.0034 \, dV$.

If D and L are in feet, gallons = $5.875 D^2L$.

Cylindrical Vessels, Tanks and Cisterns; Diameter in Feet and Inches. Area in square feet and Capacity in U. S. Gallons for one ft. in depth. *

(1 gallon = 231 cubic inches = 1 cubic foot/7.4805 = 0.13368 cubic foot.)

| | am- ter, in. | Area, square feet | Gallons, I foot | et | am- er, | square | Gallons, | et | am- er. | square | Gallon I foot |
|-----|--------------------|-------------------------|--------------------|----------|------------|--------|----------|----------|------------|------------------|------------------|
| | ın. | leet | depth | It. | in. | feet | depth | ft. | in. | feet | depth |
| I | 0 | .785 | 5.87 | 5 | 8 | 25.22 | 188 66 | 19 | 0 | 283.53 | 2120.0 |
| 1 | I | .922 | 6 89 | 5 | 9 | 25.97 | 194.25 | 19 | 3 | 201.04 | 2177.1 |
| I | 2 | 1.069 | 8.00 | 5 | IO | 26.73 | 199.92 | 19 | 6 | 298.65 | 2234 0 |
| I | 3 | 1.227 | 9.18 | 5 | II | 27.49 | 205.67 | 19 | 9 | 306.35 | 2201.7 |
| I | 4 | 1.396 | 10.44 | 6 | 0 | 28.27 | 211.51 | 20 | 0 | 314.16 | 2350.1 |
| I | 5 | 1.576 | II.79 | 6 | 3 | 30.68 | 229.50 | 20 | 3 | 322.06 | 2400.2 |
| I | | 1.767 | 13.22 | 6 | 6 | 33.18 | 248.23 | 20 | 6 | 330 06 | 2469 1 |
| I | 7 | 1.969 | 14.73 | 6 | 9 | 35 78 | 267.69 | 20 | 9 | 338.16 | 2520 (|
| I | 8 | 2.182 | 16.32 | 7 | 0 | 38.48 | 287.88 | 21 | 0 | 346.36 | 2591 |
| I | 9 | 2.405 | 17.99 | 7 | 3 | 41.28 | 308.81 | 21 | 3 | 354.66 | 2653 |
| I | IO | 2.640 | 19.75 | 7 | 6 | 44.18 | 330.48 | 21 | 6 | 363.05 | 2715.8 |
| I | II | 2.885 | 21.58 | 7 | 9 | 47.17 | 352.88 | 21 | 9 | 371.54 | 2770.3 |
| 2 | 0 | 3.142 | 23.50 | 8 | 0 | 50.27 | 376.01 | 22 | 0 | 380.13 | 2843.6 |
| 2 | I | 3.400 | 25.50 | 8 | 3 | 53.46 | 399.88 | 22 | 3 | 388.82 | 2008.6 |
| 2 | 2 | 3.687 | 27.58 | 8 | 6 | 56.75 | 424.48 | 22 | 6 | 397.61 | 2974.3 |
| 2 | 3 | 3.976 | 29.74 | 8 | 9 | 60.13 | 449.82 | 22 | 9 | 406.49 | 3040 8 |
| | 4 | 4.276 | 31.99 | 9 | 0 | 63.62 | 475.89 | 23 | 0 | 415.48 | 3108 0 |
| 2 2 | 5 | 4.587 | 34.31 | 9 | 3 | 67.20 | 502.70 | 23 | 3 | 424.56 | 3175.9 |
| 2 | | 4.909 | 36:72 | 9 | 6 | 70.88 | 530.24 | 23 | 6 | 433.74 | 3244 6 |
| 2 | 7 8 | 5.241 | 39.21 | 9 | 9 | 74.66 | 558.51 | 23 | 9 | 443.0I | 3314 0 |
| 2 | 9 | 5.585 | 41.78 | IO | 0 | 78.54 | 587.52 | 24 | 0 | 452.39 | 3384.1 |
| 2 | IO | 5.940 | 44.43 | IO | 3 | 82.52 | 617.26 | 24 | 3 | 461 86 | 3455 C |
| 2 | II | 6.305 | 47.16 | IO | 6 | 86.59 | 647.74 | 24 | 6 | 471.44 | 3526 6 |
| | 0 | 7.060 | 49.98 | IO | 9 | 90.76 | 678.95 | 24 | 9 | 481.11 | 3598.9 |
| 3 | I | 7.467 | 52.88 | 11 | 0 | 95.03 | 710.90 | 25 | 0 | 490.87 | 3072.0 |
| 3 | 2 | 7.876 | 55.86 | II | 3 | 99.40 | 743.58 | 25 | 3 | 500 74 | 3745 8 |
| 3 | 3 | 8.296 | 58.92 62.06 | II | 6 | 103.87 | 776.99 | 25 | 6 | 510 71 | 3820 3 |
| 3 | 4 | 8.727 | 65.28 | II | 9 | 108.43 | 811.14 | 25 | 9 | 520.77 | 3895 6 |
| 3 | 5 | 9.168 | 68.58 | 12 12 | 0 | 113.10 | 846.03 | 26 | 0 | 530 93 | 3971 6 |
| 3 | 6 | 9.621 | 71.97 | 12 | 3 | 117 86 | 881.65 | 26 | 3 | 541 19 | 4048 4 |
| 3 | 7 | 10.085 | 75.44 | 12 | | 122 72 | 918.00 | 26 | 6 | 551 55 | 4125 9 |
| 3 | 8 | 10.550 | 78.00 | 13 | 9 | 127 68 | 955.09 | 26 | 9 | 562 00 | 4204.I |
| 3 | 9 | 11.045 | 82.62 | 13 | | 132.73 | 992.91 | 27 | 0 | 572.50 | 4283 0 |
| 3 | IO | 11.541 | 86.33 | 13 | 6 | 137.89 | 1031.5 | 27 | 3 | 583.21 | 4362.7 |
| 3 | II | 12.048 | 90.13 | 13 | 9 | 143.14 | 1070 8 | 27 | 6 | 593 96 | 4443.1 |
| 4 | 0 | 12.566 | 94.00 | 14 | 0 | 148.49 | 1110.8 | 27 | 9 | 604 81 | 4524-3 |
| 4 | I | 13.005 | 97.96 | 14 | 3 | 150.48 | 1151.5 | 28 | 0 | 615.75 | 4606.2 |
| 4 | 2 | 13.635 | 102.00 | 14 | 6 | 165.13 | 1193.0 | 28 | 3 | 626 80 | 4688 8 |
| 4 | 3 | 14.186 | 106.12 | 14 | 9 | 170 87 | 1235.3 | 28 28 | 6 | 637 94 | 4772 I |
| 4 | 4 | 14.748 | 110.32 | 15 | 0 | 176.71 | 1321.0 | 20 | 9 | 649.18 | 4856.2 |
| 4 | 5 | 15.321 | 114.61 | 15 | 3 | 182 65 | 1366.4 | 20 | | 660 52 | 4941.0 |
| 4 | 6 | 15 00 | 118.97 | 15 | 6 | 188.60 | 1411 5 | 29 | 3 6 | 683 49 | 5026 6 |
| 4 | 7 | 16 50 | 123.42 | 15 | 9 | 194.83 | 1457.4 | 29 | 9 | | 5112.9 |
| 4 | 8 | 17 10 | 127.95 | 16 | 0 | 201.06 | 1504.1 | 30 | 0 | 695.13 706.86 | 5199.9 |
| 4 | 9 | 17 72 | 132 56 | 16 | 3 | 207.30 | 1551.4 | 30 | 3 | 718 60 | 5287.7 |
| 4 | IO | 18 35 | 137 25 | 16 | 6 | 213 82 | 1599.5 | 30 | 6 | | 5376 2 |
| 4 | II | 18 99 | 142 02 | 16 | 0 | 220 35 | 1648 4 | 30 | 0 | 730 62 742 64 | 5465.4 |
| 5 | 0 | 19 63 | 146 88 | 17 | 0 | 226 98 | 1697.9 | 31 | 0 | | 5555.4 |
| 5 | I | 20 20 | 151 82 | 17 | 3 | 233 71 | 1748 2 | 31 | 3 | 754 77 766 99 | |
| 5 | 2 | 20 07 | 156 83 | 17 | 6 | 240 53 | 1799 3 | 31 | 6 | 779.31 | 5737.5 5829.7 |
| 5 | 3 | 21 65 | 161 93 | 17 | 0 | 247.45 | 1851.1 | 31 | 9 | 701.73 | 5922.6 |
| 5 | 4 | 22 34 | 167.12 | 18 | 0 | 254 47 | 1903.6 | 32 | 0 | 804 25 | 6016.2 |
| 5 | 5 | 23 04 | 172 38 | 18 | 3 | 261 59 | 1956.8 | 32 | 3 | 816 86 | 6110.6 |
| 5 | 6 | 23 76 | 177 72 | 18 | 6 | 268 80 | 2010 8 | 32 | 6 | 820 58 | 6205 7 |
| 5 | 7 | 24 48 | 183 15 | 18 | 0 | 276 12 | 2065 5 | 32 | 9 | 842 39 | 6301 5 |

Loss in Water Pressure in Pipes, due to Friction For Each 100 Feet of Length in Different Size Clean Iron Pipes, Discharging Given Quantities of Water per Minute Also Velocity of Flow in Pipe, in Feet per Second

| geq | Gallons Discharg per Minute | 250 | 35 4 6 | 50 100 | 125 150 175 | 200 | 350 400 450 | 500 750 1000 | 1250 |
|--------|--|----------------------|----------------------|----------------------|----------------------|-------------------------|----------------------|----------------------|-------|
| H | Priction Loss sbanoq ai | | | | | 0.07 | | 0.25 | 1.46 |
| 8 INCH | Veloc. in Pipe in Feet per Second | | | | | 1.59 | | 3.19 4.79 6.38 | 7.97 |
| НО | Priction Loss sbanoq ni | | | 0.05 | 0.10 | 0.17 0.26 0.37 | 0.50 | 0.96 2.21 3.88 | |
| 9 ING | Veloc. in Pipe in Feet per Second | | | 1.13 | 1.70 | 2.27 2.84 3.40 | 3.97 4.54 5.11 | 5.67 8.51 11.3 | : |
| CH | Friction Loss sbanoq ni | | | 0.09 | 0.69 | 1.22 1.89 2.66 | 3.65 4.73 6.01 | 7.43 | |
| 4 INC | Veloc. in Pipe in Feet per Second | | | 1.28 | 3.83 | 5.11 6.39 7.66 | 8.94 10.2 11.5 | 12.8 | |
| . H(| Friction Loss shanoq ni | 0.10 | | 0.35 | 1.99 2.85 3.85 | 5.02 7.76 11.2 | 15.2 19.5 25.0 | 30.8 | : |
| 3 INCH | Veloc. in Pipe in Feet per Second | 1.13 | | 2.27 3.40 4.54 | 5.67 6.81 7.94 | 9.08 11.3 13.6 | 15.9 18.2 20.4 | 22.7 | |
| CH | Friction Loss in Pounds | 0.21 | | 0.81 1.80 3.20 | 4.89 7.00 9.46 | 12.48 19.66 28.06 | | | |
| 2½ INC | Veloc. in Pipe in Feet per Second | 1.63 | | 3.26 4.90 6.53 | 8.16 9.80 11.4 | 13.1 16.3 19.6 | | | |
| H | Friction Loss in Pounds | 0.42 | 1.60 | 2.44 5.32 9.46 | 14.9 21.2 28.1 | 37.5 | | | : : : |
| 2 INCH | Veloc. in Pipe in Feet per Second | 2.04 | 4.09 | 5.11 7.66 10.2 | 12.8 15.3 17.1 | 20.4 | | | |
| INCH | Friction Loss sbanod ni | 1.66 2.62 3.75 | 5.05 6.52 8.15 | 10.0 22.4 39.0 | | | | | : |
| 1½ IN | Veloc. in Pipe in Feet per Second | 3.63 4.54 5.45 | 6.36 7.26 8.17 | 9.08 13.6 18.2 | | | | | |
| INCH | Friction Loss in Pounds | 4.07 6.40 9.15 | 12.4 16.1 20.2 | 24.9 | | | | | : |
| 1¼ IN | Veloc. in Pipe in Feet per Second | 5.22 6.53 7.84 | 9.14 10.4 11.7 | 13.1 | | | | | |
| H | Friction Loss in Pounds | 12.3 19.0 27.5 | 37.0 | | | | | | |
| 1 INCH | Veloc. in Pipe in Feet! per Second | 8.17 10.2 12.3 | 14.3 | | | | | | |
| CH | Friction Loss spanoffni | 50.4 | | | | | | | |
| % INCH | Veloc. in Pipe in Feet per Second | 14.5 | | | | | | | |
| | Gallons Discha | 20 30 | 35 40 45 | 50 75 100 | 125 150 175 | 200 250 300 | 350 400 450 | 500 750 1000 | 1250 |

Loss of Air Pressure in Pipes, due to Friction; Applies to all Pressures and for Volumes up to 3000 Cu. Ft. of Free Air

| Cubic Feet of Free Air per Minute | Divide the number corresponding to the diameter of the pipe and volume in cu. ft. of free air by the ratio of compressions from free air. The result is the loss in pounds per sq. inch in 1000 feet of pipe. *See example. Diameter of Pipe in Inches | | | | | | | | | | | | |
|--|---|--|---|--|---|---|--|--|---|--|--|--|--|
| FF FF er | 1/ | 9/ | | 1 | 1 | 1 | 1 | 1 2 | 91/ | | | | |
| 054 | 1/2 | 3/4 | 1 | 11/4 | $1\frac{1}{2}$ | 2 | $2\frac{1}{2}$ | 3 | 31/2 . | | | | |
| 6 12 24 48 75 100 125 150 200 250 300 | 43.5 173. 753. | 5.7 22.8 91. 362. | 1.4 5.3 21. 85. 200. 373. | .4 1.5 5.9 23.5 57. 92. 163. 230. 368. | 2.2 8.5 21. 37.5 59. 83.5 150. 216. 338. | 2. 4.9 8.7 13.6 19.5 35. 54.5 79. | 1.6 2.7 4.4 6.3 10.8 17.7 25.3 | 2.3 3.7 6.4 9.3 | 2.9 | | | | |
| | 2 | 21/2 | 3 | 31/2 | 4 | 41/2 | 5 | 6 | 7 | | | | |
| 350 400 450 500 600 700 800 900 1000 1200 1400 1800 2100 2400 2700 3000 | 105. 140. 174. 216. 316. | 33. 45. 55. 68. 100. 133. 172. 219. 272. 400. | 12.7 14.8 20.7 26. 37. 51. 59. 83. 104. 204. 236. 332. | 5.5 7.1 9.4 11.1 16.6 21.8 29. 38. 44. 67. 87. 114. 150. 196. 266. | 2.7 3.6 4.1 5.6 7.3 10.9 14.3 16.5 22. 29. 44. 57. 66. 98. 117. 149. 201. | 1.9 2.3 2.9 4. 5.8 7.5 9.2 11.7 16. 23. 30. 35. 52. 65. 82. | 1.7 2.5 3.4 4.4 5.4 6.8 9.6 14. 18. 22. 30. 38. 49. 60. | 1.8 2.2 2.8 3.9 5.5 7.1 8.8 12.5 15.5 19.5 24. | 1.4 1.9 2.6 3.1 4.3 5.9 7.7 9.8 12. | | | | |

What is the loss of pressure and the terminal pressure with 350 cu. ft. of free air compressed to 90 lbs.? Initial gauge pressure passing through 500 ft. of 2 in. pipe.

From the table it will be seen that the number corresponding to the diameter of

pipe and the volume mentioned is 105.

 $\frac{90 + 14.7}{2} = 7.1$ Atmospheres or Compressions.

14.7 $\frac{14.7}{1000'} = 2$ $\frac{105.}{2} = 52.5$ $\frac{52.5}{7.1} = 7.4 \text{ Lbs. Loss.}$

90 Lbs. Initial -7.4 Lbs. Loss = 82.6 Lbs. Terminal pressure. This table is adapted from Elmo G. Harris' Formula in Compressed Air. McGraw-Hill Book Co., 1910. $f = c \frac{1}{d^5} \times \frac{(Va)^2}{r}$

f=Loss of pressure per sq. inch.
c=An experimental coefficient.
l=Length of pipe in feet.
d=Diameter of pipe in inches.
Va=Cubic feet of free air passing per second.
r=Ratio of compressions from free air.

Values for "C" Used
d=in. ½ ¾ 1 1¼ 1½ 2 2½ 3 3 ½ 4 4½ 5 6 7
c= 1.47 1.21 1.35 1.16 1.02 1 .95 .9 .86 .825 .775 .715 .825 .8

Horsepower (Theoretical) Required to Compress 100 Cu. Ft. Free Air to Various Pressures

| | | m . C | Saving of Two-Stage Over Single- Stage Compression | | | | |
|---|--|---|--|--|--|--|--|
| Gauge Pressure | Single-Stage | Two-Stage | Horsepower | Per Cent | | | |
| 5 10 15 20 25 30 35 40 45 50 55 60 65 70 75 80 85 90 95 100 110 120 130 140 150 160 170 180 190 200 210 220 230 240 250 260 270 280 290 300 350 400 400 400 400 400 400 400 400 400 4 | 1.97 3.61 5.02 6.28 7.44 8.45 9.41 10.30 11.13 11.92 12.67 13.37 14.05 14.70 15.32 15.91 16.48 17.04 17.57 18.09 19.08 20.01 20.90 21.74 22.55 23.32 24.06 24.77 25.46 26.12 | 10.65 11.25 11.81 12.34 12.84 13.32 13.77 14.21 14.63 15.03 15.42 16.15 16.83 17.46 18.07 18.64 19.26 19.78 20.27 20.74 21.19 21.54 21.96 22.37 22.76 23.03 23.28 23.84 24.19 24.53 24.85 26.35 27.65 28.85 29.97 | 1.28 1.42 1.57 1.71 1.85 2.00 2.13 2.27 2.41 2.54 2.67 2.93 3.18 3.43 3.67 3.91 4.06 4.29 4.51 4.70 4.93 | 10.70 11.22 11.72 12.18 12.61 13.04 13.77 14.12 14.45 14.77 15.36 15.90 16.42 16.89 17.33 17.40 17.80 18.18 18.46 18.88 | | | |

Flow of Air Through an Orifice in Cubic Feet of Free Air per Minute

Flowing from a Round Hole in a Receiver into the Atmosphere

These quantities are correct only for an orifice with countersunk edges and are not correct for a hole with straight edges.*

RECEIVER GAUGE PRESSURE

| Diameter of Orifice Inches | 2 Pounds | 5 Pounds | 10 Pounds | 15 Pounds | 20 Pounds | 25 Pounds | 30 Pounds | 35 Pounds | 40 Pounds | 45 Pounds | 50 Pounds | 60 Pounds | 70 Pounds | 80 Pounds | 90 Pounds | 100 Pounds | 125 Pounds |
|----------------------------------|-------------|-------------|--------------|--------------|--------------|--------------|--------------|--------------|--------------|--------------|--------------|--------------|--------------|--------------|--------------|---------------|---------------|
| 1 64 | .038 | .0597 | .0842 | .103 | .119 | .133 | .156 | .173 | .19 | .208 | .225 | .26 | .295 | .33 | .364 | .40 | .486 |
| $\frac{1}{32}$ | .153 | .242 | .342 | .418 | .485 | .54 | .632 | .71 | .77 | .843 | .914 | 1.05 | 1.19 | 1.33 | 1.47 | 1.61 | 1.97 |
| 16 | .647 | .965 | 1.36 | 1.67 | 1.93 | 2.16 | 2.52 | 2.80 | 3.07 | 3.36 | 3.64 | 4.2 | 4.76 | 5.32 | 5.87 | 6.45 | 7.85 |
| 1/8 | 2.435 | 3.86 | 5.45 | 6.65 | 7.7 | 8.6 | 10. | 11.2 | 12.27 | 13.4 | 14.50 | 10.8 | 19. | 21.2 | 23.50 | 25.8 | 31.4 |
| 1/4 | 9.74 | 15.40 | 21.8 | 26.70 | 30.8 | 34.5 | 40. | 44.7 | 49.09 | 53.8 | 58.2 | 67. | 76. | 85. | 94. | 103. | 125. |
| 3/8 | 21.95 | 34.60 | 49. | 60. | 69. | 77. | 90. | 100. | 110.45 | 121. | 130. | 151. | 171. | 191. | 211. | 231. | 282. |
| 1/2 | 39.00 | 61.60 | 87. | 107. | 123. | 138. | 161. | 179. | 196.35 | 215. | 232. | 268. | 304. | 340. | 376. | 412. | 502. |
| 5/8 | 61.00 | 96.50 | 136. | 167. | 193. | 216. | 252. | 280. | 306.80 | 336. | 364. | 420. | 476. | 532. | 587. | 645. | 785. |
| 3/4 | 87.60 | 133. | 196. | 240. | 277. | 310. | 362. | 400. | 441.79 | 482. | 522. | 604. | 685. | 765. | 843. | 925. | |
| 7/8 | 119.50 | 189. | 267. | 326. | 378. | 422. | 493. | 550. | 601.32 | 658. | 710. | 622. | 930. | 1004. | | | |
| 1 | 156. | 247. | 350. | 427. | 494. | 550. | 645. | 715. | 785.40 | 860. | 930. | | | | | | |
| 11/4 | 242. | 384. | 543. | 665. | 770. | 860. | 1000. | | | | | | | | | | |
| 11/2 | 350. | 550. | 780. | 960. | | | | | | | | | | | | | |
| 2 | 625. | 985. | | | | | | | | | | | | | | | |

^{*} See Bulletin 58-L. page 18.

Cylinder Piston Displacements in Cubic Feet per Revolution, of Double Acting Cylinders

| Diam. in Inches | | | | | | | STROE | KE IN I | NCHES | | | | | | | Diam. |
|----------------------------|--------------------------------------|-----------------------------------|---|---|---|---|---|---|---|---|---|---|---|---|---|----------------------------|
| | 6 | 7 | 8 | 10 | 12 | 14 | 16 | 1 18 | 20 | 1 22 | 24 | 30 | 36 | 1 42 | 1 48 | |
| 6 7 8 9 10 | .195 .267 .349 .442 .545 | .229 311 .407 515 636 | .261 .356 .465 588 727 | 327 445 .581 736 909 | 392 534 698 .883 1 090 | .458 624 814 1.030 1.272 | 523 712 930 1.177 1 454 | 1.04 1.32 1.63 | 1.16 1.47 1.81 | 1.27 1.61 2.00 | 1.39 1.76 2.18 | 1.74 2.20 2.72 | 3.27 | | | 6 7 8 9 10 |
| 11 12 13 14 15 | .659 785 | 769 916 1 075 1.247 | 879 1.047 1.228 1 425 1 636 | 1 099 1 308 1 536 1 781 2 045 | 1 319 1.571 1.843 2 137 2 454 | 1 539 1.832 2.150 2.494 2.803 | 1.759 2.094 2.457 2.850 3.272 | 1.97 2.35 2.76 3.20 3.68 | 2.19 2.61 3.07 3.56 4.09 | 2.41 2.87 3.37 3.91 4.49 | 2.63 3.14 3.68 4.27 4.90 | 3.29 3.92 4.60 5.34 6.13 | 3.95 4.71 5.53 6.41 7.36 | 5.49 6.45 7 48 8.58 | 8.55 9.81 | 11 12 13 14 15 |
| 16 17 18 19 20 | | | 1.861 | 2.327 2.629 2.945 | 2.792 3.152 3.534 3.937 4.363 | 3 257 3 677 4 123 4.593 5.090 | 3 723 4.203 4.712 5.250 5 817 | 4 18 4.72 5.30 5 90 6.54 | 4.65 5 25 5 89 6.56 7 27 | 5.12 5.77 6.47 7.21 7 99 | 5.58 6.30 7.06 7.87 8.72 | 6.98 7.88 8.83 9.84 10.90 | 8.37 9.45 10.60 11.81 13.08 | 9.77 11.03 12.36 13.78 15.27 | 11 16 12 60 14 13 15 75 17 45 | 16 17 18 19 20 |
| 21 22 23 24 25 | | | | | | 5 612 6 159 | 6 413 7 037 7 693 8 377 9 090 | 7 21 7 91 8 65 9 42 10 22 | 8 01 8.79 9 61 10 47 11 36 | 8.81 9 67 10 57 11 52 12.49 | 9 62 10 55 11.54 12.56 13 63 | 12 02 13.19 14 42 15.70 17 04 | 14 43 15.83 17 31 18.84 20.44 | 16.83 18.47 20 19 21 98 23 85 | 19 24 21 11 23.08 25.12 27 26 | 21 22 23 24 25 |
| 26 27 28 29 30 | | | | | | | 9 830 10 602 11 402 | 11 06 11 92 12 82 13 75 14 72 | 12 29 13 25 14 25 15 28 16 36 | 13 51 14.57 15 67 16 81 17 99 | 14 74 15 90 17 10 18 34 19 63 | 18.43 19.88 21.38 22.93 24.54 | 22 12 23 85 25 65 27 51 29 44 | 25 80 27 83 29 93 32 10 34.35 | 29 49 31.80 34 20 36 68 39 26 | 26 27 28 29 30 |
| 31 32 33 34 35 | | | | | | | | 15 72 16 75 | 17 47 18 61 | 19 21 20 47 21 77 23 11 | 20 96 22 34 23 75 25 22 26 72 | 26 20 27 92 29 69 31 52 33 40 | 31 44 33 51 35 63 37 83 40 08 | 36 68 39 09 41 57 44.13 46 76 | 41 92 44 68 47 51 50 44 53 45 | 31 32 33 34 35 |
| 36 37 38 39 40 | | | | | | | | | | | 28 27 | 35 34 37.33 39 37 41 47 43 63 | 42 40 44 79 47 25 49.77 52 35 | 49 47 52 26 55 12 58 06 61 08 | 56 54 59 72 63 00 66 36 69 80 | 36 37 38 39 40 |
| 42 | | | | | | | | | | | | | | 67.34 73.89 | 76 96 84 46 | 42 |

Sullivan Machinery Company

122 South Michigan Ave., Chicago Works: Claremont, N. H., Chicago, Illinois

| AMSTERDAM, HOLLAND | | | | Heerengracht, No. 141 |
|-----------------------|---|---|------|------------------------|
| BARRE, VERMONT | | | | |
| Boston, Mass | | | | . 185 Devonshire St. |
| Butte, Mont | | | | . 48 E. Broadway |
| CHRISTIANIA, NORWAY | | | | . Toldbodgaten 8b |
| CLAREMONT, N. H. | | | | Main St. |
| COBALT, ONT | | • | | |
| DENVER, COLO | | • | | . 837 Equitable Bldg. |
| EL PASO, TEXAS | | • | | 511 Mills Bldg. |
| HAVANA, CUBA | | • | | |
| | | | | . Teniente Rey, 11 |
| HUNTINGTON, W. VA . | | | | 641 Court St. |
| Ishpeming, Mich | | | | 14h J W-11 C+- |
| Joplin, Mo | | • | | . 4th and Wall Sts. |
| Juneau, Alaska | | | | Forrest Bldg. |
| KNOXVILLE, TENN | | | | Houston Bldg. |
| London, England . | | | | Salisbury House |
| Manila, P. I. | | | | . Calle Echague, 64 |
| Nelson, B. C | | | | |
| New York | | | | 30 Church St. |
| Paris, France | | | | . 18 Ave. Parmentier |
| Petrograd, Russia . | | B | olsl | haia Koniushennaia, 29 |
| PITTSBURGH, PA | | | | Farmers Bank Bldg. |
| SALT LAKE CITY, UTAH | | | | Kearns Bldg. |
| SAN FRANCISCO, CALIF. | 1 | | | 461 Market St. |
| SANTIAGO, CHILE | | | | . Teatinos, 349 |
| SHANGHAI, CHINA | | | 4 | Yuen Ming Yuen Road |
| SPOKANE, WASH | | | | Hutton Bldg. |
| St. Louis, Mo | | | | . Railway Exchange |
| SYDNEY, AUSTRALIA . | | | | Australasia Chambers |
| TIEN TSIN, CHINA . | | | | |
| Tokyo, Japan | | | | |
| TORONTO, CANADA . | | | 1 | 37 Colborne St. |
| TURIN, ITALY | | | | Corso San Martino, 4 |
| VANCOUVER, B. C | | | | Granville St. Bridge |
| VARCOUVER, D. C | | | | Orani vine ber Bridge |

Manufacturers of

Diamond Prospecting Core Drills; Coal Cutters, Air or Electric; Air Compressors for all purposes; Air Lift Pumps; Rock Drills and Hammer Drills for excavating rock; Drill Sharpeners; Automatic Cross-over Car Dumps; Hoisting Engines for deep mines; Quarrying Machinery.

Cable Addresses: "Diamond, Chicago," "Payotto, Christiania," "Mikewal, London," "Sullimaco, Paris," "Eidorb, Sydney," "Collegium, Petrograd," "Dolivan, Santiago" (Chile.)

Codes Used: "Al," "ABC," "Commercial Directory," "Fraser & Chalmers," "General," "Liebers" (5 letter), "Western Union."

